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Great Lakes Water Authority (GLWA)
Michigan, USA

- Manager for GLWA's systems control center, leading the operations and maintenance for GLWA's water distribution and wastewater collection systems.
- A process control engineer at PCI Detroit Co., worked on a multi-year project to modernize DWSD's water and wastewater systems with control and SCADA solution.
- College of Engineering, Wayne State University, USA, 2000.

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- Operational engineer for GLWA's systems control center, working on the operations, maintenance and optimization for GLWA's water system.
- A project engineer at Alfred & Benesch Co., worked on hydraulic modeling, water master plan, pipeline and pumping station design projects.
- College of Engineering, Wayne State University, USA, 1994.

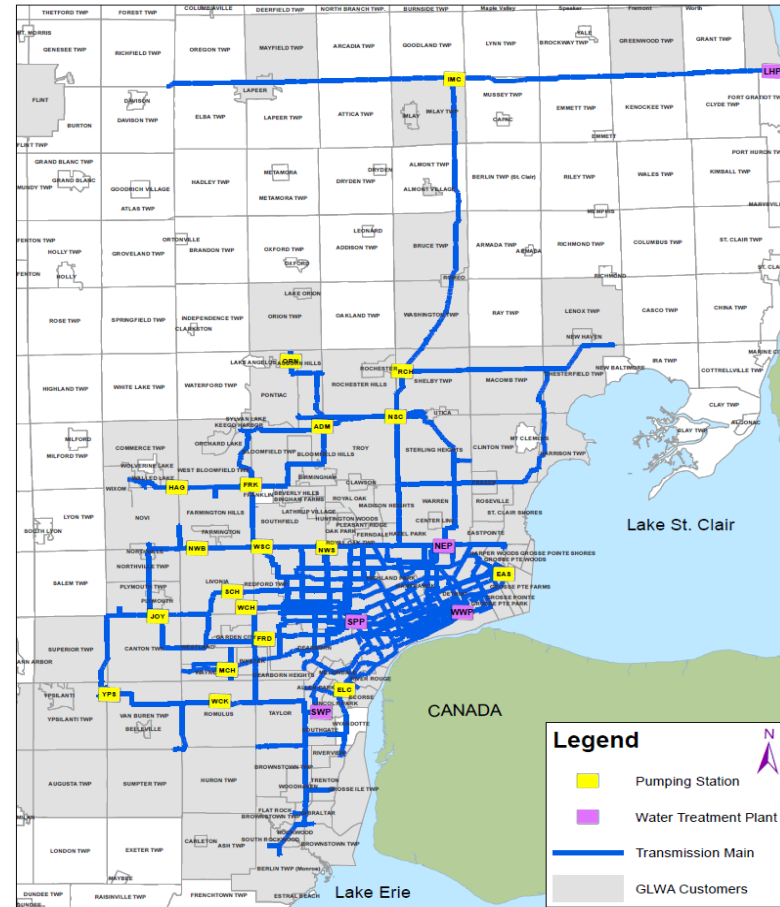
Real-Time Pump Efficiency Evaluation Program for A Large Water System

Biren Sapia & Steven Jin
Great Lakes Water Authority



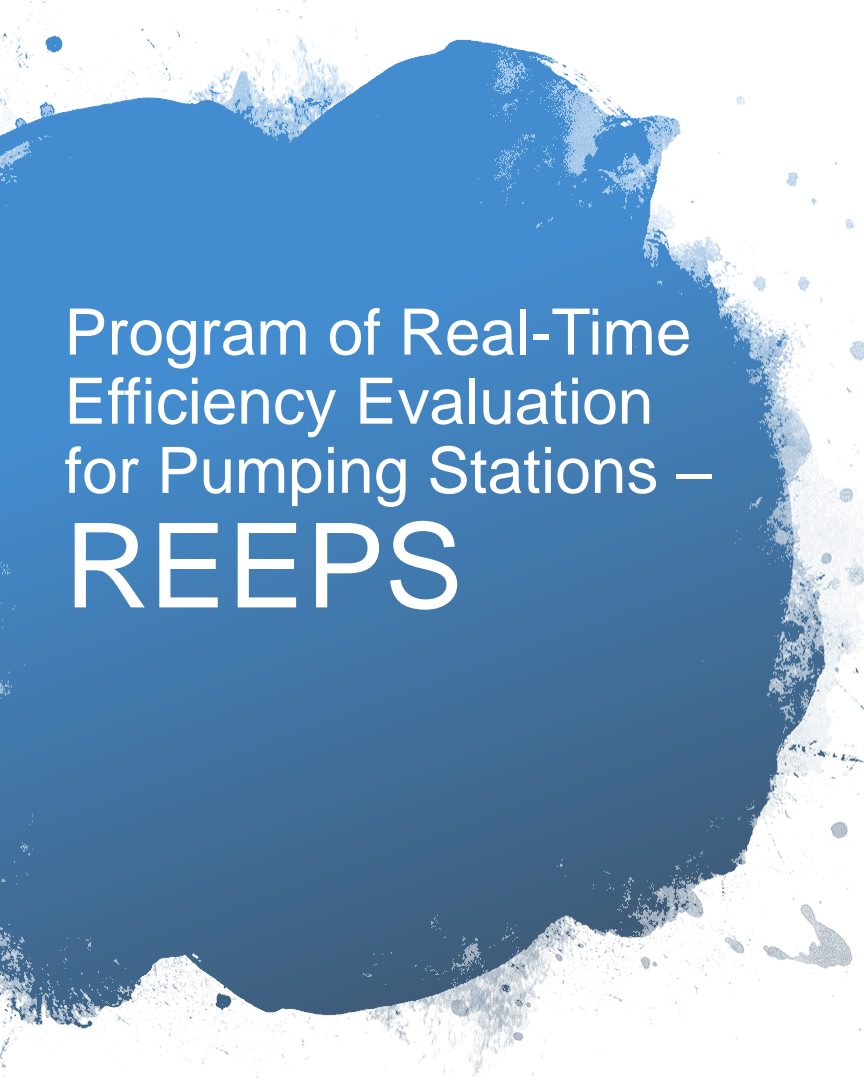
GLWA's Water System

- Serving 3.8 million people in 112 communities
- 5 treatment plants
- 19 pumping stations
- 816 miles of transmission main



GLWA's Systems Control Center





Program of Real-Time Efficiency Evaluation for Pumping Stations – REEPS

- GLWA pays over \$20M annual electricity costs for pumping water to the customers
- To analyze pump efficiency, GLWA developed the REEPS program
- REEPS would evaluate real-time efficiency for each of the 120 booster pumps and 54 high lift pumps
- REEPS would find ways to improve pumping efficiency for individual pumps and systematic operations

REEPS Method

Evaluating Pump Efficiency:

- $\eta_p = P_{hyd} / P_b$
- $P_b = P_m \times \eta_m$
- Install a power monitoring meter for each pump (P_m)

Where:

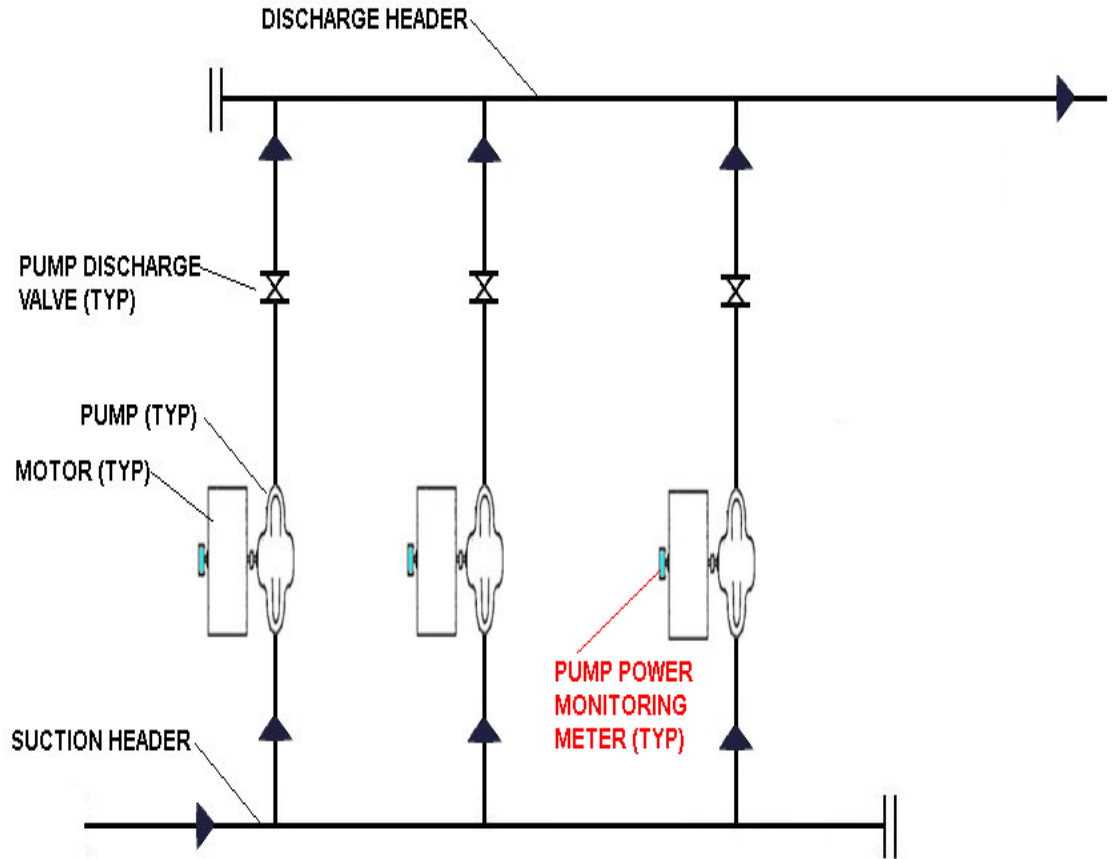
η_p = the pump efficiency

P_{hyd} = the pump hydraulic power in kilowatts

P_b = the pump input (brake) power in kilowatts

P_m = the motor power in kilowatts

η_m = the motor efficiency
(assuming 93%)



REEPS Method

Hydraulic Power*: P = power in hp

$$P = hQ/360$$

*source: Quora.com

h = pump head in ft

Q = flow-rate in gpm

$$P_{\text{hyd}} = 0.131 \times Q \times H_{\text{tdh}}$$

A flow meter is not fitting in a pump system due to its lay length.

Pump head curve formulation*:

$$H_{\text{tdh}} = H_{\text{st}} - a Q^b$$

*source: EPANET Users Manual

Where:

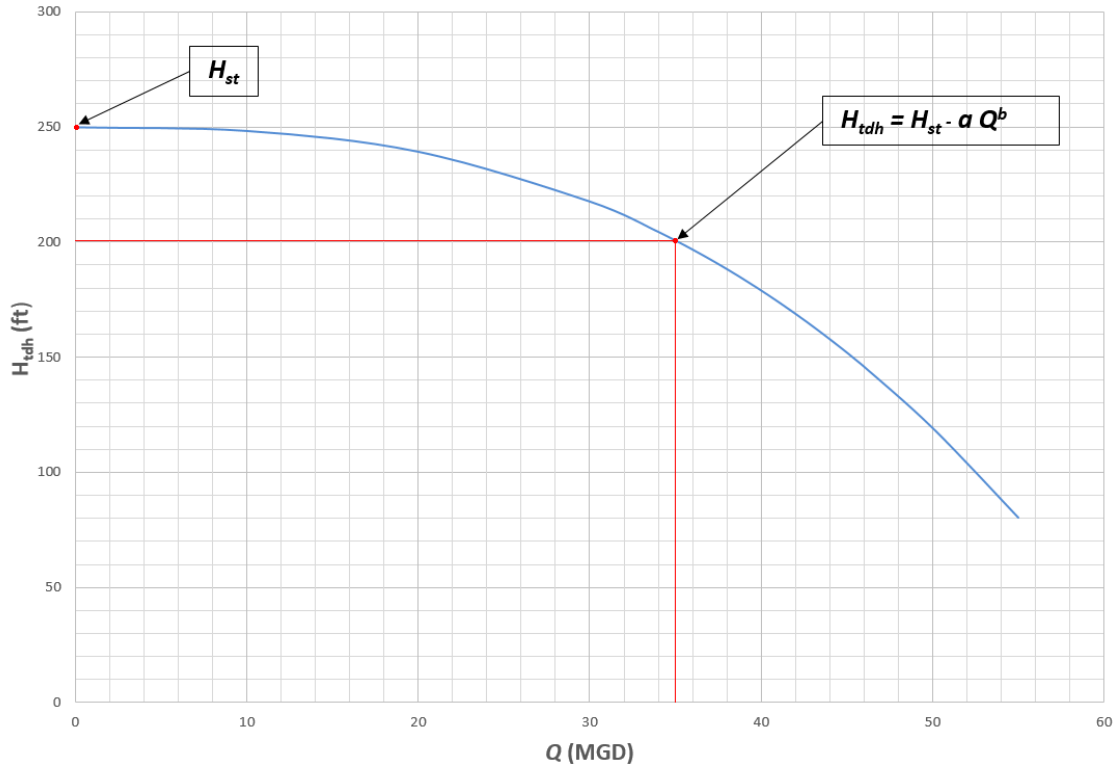
Q = the pump flow rate in MGD

H_{tdh} = pump total head in feet

H_{st} = pump shut-off head in feet

a = pump resistance coefficient

b = pump flow exponent



REEPS Method

Compute minor friction loss:

$$V = Q / A = Q / (\pi D^2 / 4)$$

$$H_{ml} = K V^2 / 2g$$
$$= 0.011 \times K \times Q^2 / D^4$$

Where:

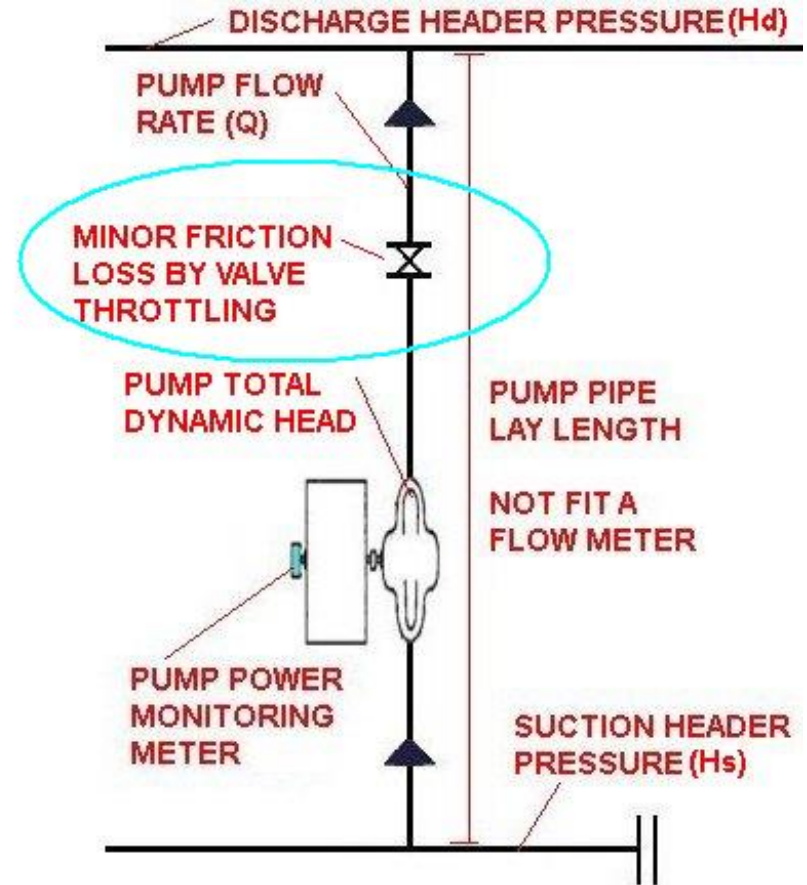
V = flow velocity in feet per second

A = the section area of pipe downstream valve (ft²)

D = the diameter of pipe downstream valve in feet

K = minor friction loss coefficient for discharge valve (based on manufacturer's valve curve data)

g = gravity acceleration (32.2 ft/s²)



Manufacturer's Valve Curve Sample

$$C_v = 29.8 D_v^2 \sqrt{\frac{1}{K}}$$

Table for valve flow coefficient values

| VALVE SIZE | PLUG ANGLE IN DEGREES FROM CLOSED | | | | | | | | |
|-----------------------|-----------------------------------|--------|-------|-------|-------|-------|------|------|------|
| | 90 | 80 | 70 | 60 | 50 | 40 | 30 | 20 | 10 |
| RESISTANCE FACTOR "K" | | .46 | 1.9 | 5.2 | 13 | 31 | 74 | 180 | 580 |
| 6 | 3596 | 1566 | 826 | 472 | 300 | 193 | 123 | 78 | 44 |
| 8 | 7423 | 2784 | 1468 | 839 | 534 | 343 | 219 | 139 | 78 |
| 10 | 12165 | 4350 | 2294 | 1311 | 834 | 536 | 342 | 217 | 122 |
| 12 | 18466 | 6265 | 3304 | 1888 | 1201 | 772 | 493 | 313 | 176 |
| 14 | 25613 | 8527 | 4497 | 2567 | 1635 | 1051 | 671 | 426 | 239 |
| 16 | 34820 | 11138 | 5874 | 3356 | 2136 | 1373 | 877 | 556 | 312 |
| 18 | 45515 | 14096 | 7434 | 4248 | 2703 | 1738 | 1110 | 704 | 395 |
| 20 | 56191 | 17403 | 9178 | 5244 | 3337 | 2145 | 1370 | 870 | 488 |
| 24 | 80915 | 25060 | 13216 | 7552 | 4806 | 3089 | 1974 | 1253 | 703 |
| 30 | 134100 | 39157 | 20651 | 11880 | 7509 | 4827 | 3084 | 1957 | 1100 |
| 36 | 200780 | 56386 | 29738 | 16993 | 10813 | 6951 | 4441 | 2819 | 1583 |
| 42 | 277053 | 76748 | 40476 | 23129 | 14718 | 9462 | 6045 | 3837 | 2155 |
| 48 | 372356 | 100242 | 52867 | 30210 | 19224 | 12358 | 7895 | 5012 | 2815 |



A 36-inch Cone Valve of GLWA

Source: <http://literature.puertoricosupplier.com/003/QG2286.pdf>

REEPS Method

Bernoulli equation for a pump system:

$$\frac{P_1}{\rho g} + \frac{V_1^2}{2g} + z_1 = \frac{P_2}{\rho g} + \frac{V_2^2}{2g} + z_2 + h_{friction} - h_{pump}$$

When $V_1 = V_2$, then

$$H_{tdh} = (H_d - H_s) + H_{ml} \dots \dots \dots \text{Eq-A}$$

Eq-A can be changed as follows and is used to solve for Q:

$$(H_{st} - a Q^b) = (H_d - H_s) + (0.011 K Q^2 D^{-4})$$

Eq-A becomes

$$f(Q) = (H_d - H_s) + (0.011 K Q^2 D^{-4}) - (H_{st} - a Q^b) = 0$$

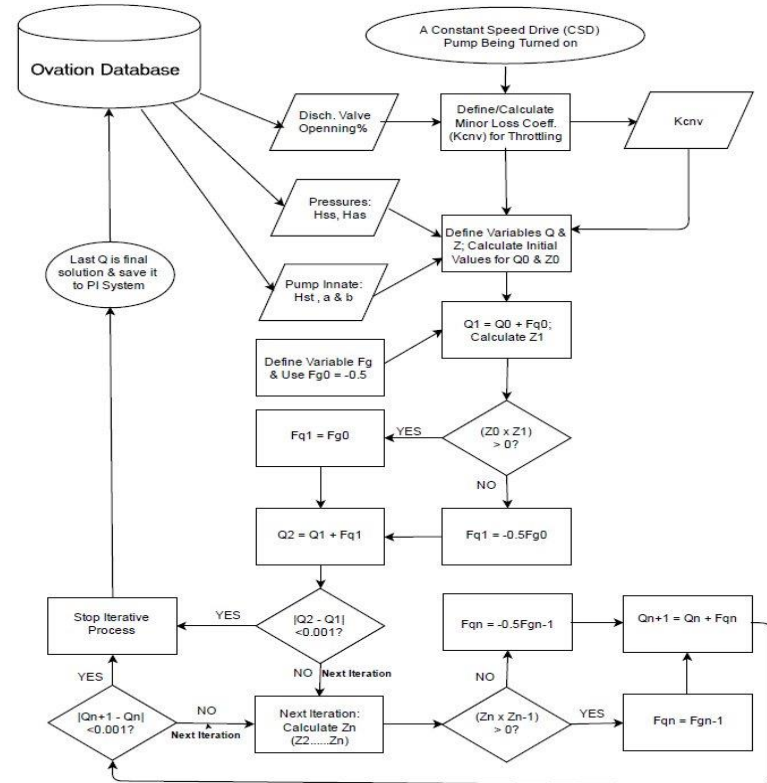
Where:

H_{ml} = minor friction loss of a pump system in feet

H_d = the elevation of station discharge head (ft)

H_s = the elevation of station suction head (ft)

An iterative algorithm to solve Eq-A



REEPS Method

When the valve curve data is unavailable:

A differential pressure transmitter (DPT) is installed for each pump.

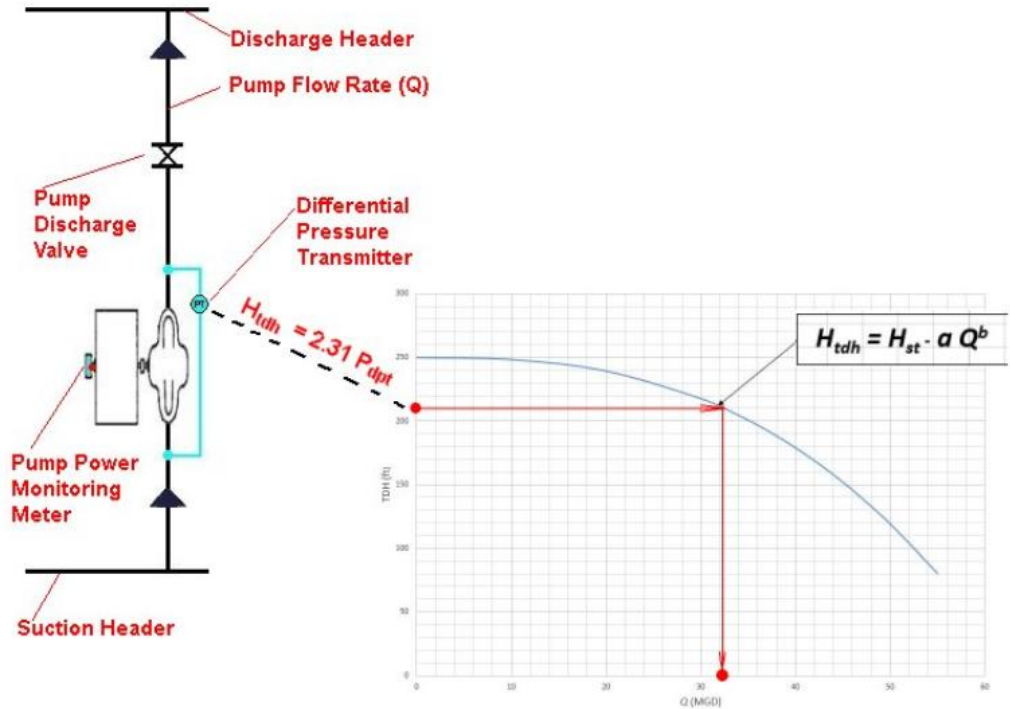
$$H_{tdh} = 2.31P_{dpt}$$

$$Q = ((H_{st} - H_{tdh}) / a)^{(1/b)}$$

$$= ((H_{st} - 2.31P_{dpt}) / a)^{(1/b)}$$

Where:

P_{dpt} = the pressure differential (psi) measured by a DPT



REEPS Method

Affinity Laws for VFD Pumps

Flow is proportional to speed

$$Q_p / Q_f = N_p / N_f = F_s$$

Pressure is proportional the square of speed

$$H_{thp} / H_{thf} = (N_p / N_f)^2 = F_s^2$$

Where:

Q_p = the partial speed flow rate for a VFD pump in MGD

Q_f = full speed flow rate for a VFD pump in MGD

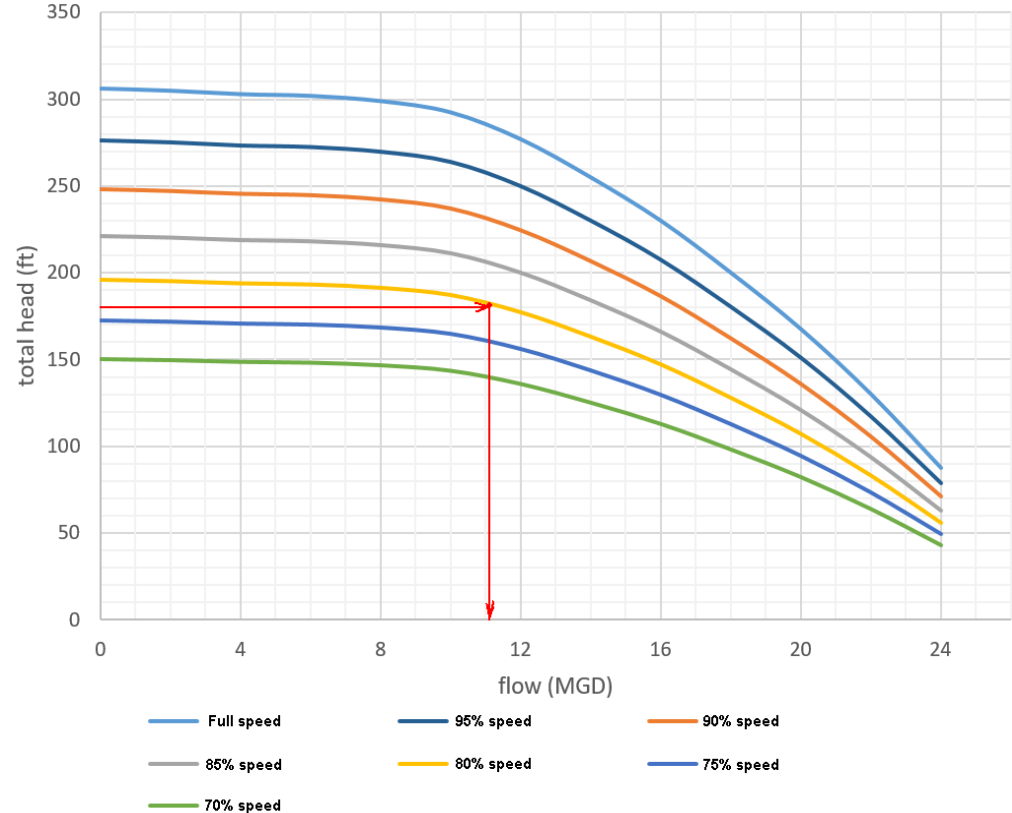
N_p = partial pump speed in RPM

N_f = full speed for a VFD pump in RPM

F_s = factor of speed (N_p / N_f)

H_{thp} = pump head under a partial speed in feet

H_{thf} = pump head under full speed in feet



REEPS Method

VFD Pump: the discharge valve is open.

Pump head curve formula:

$$H_{tdh} = H_{st} - a Q^b$$

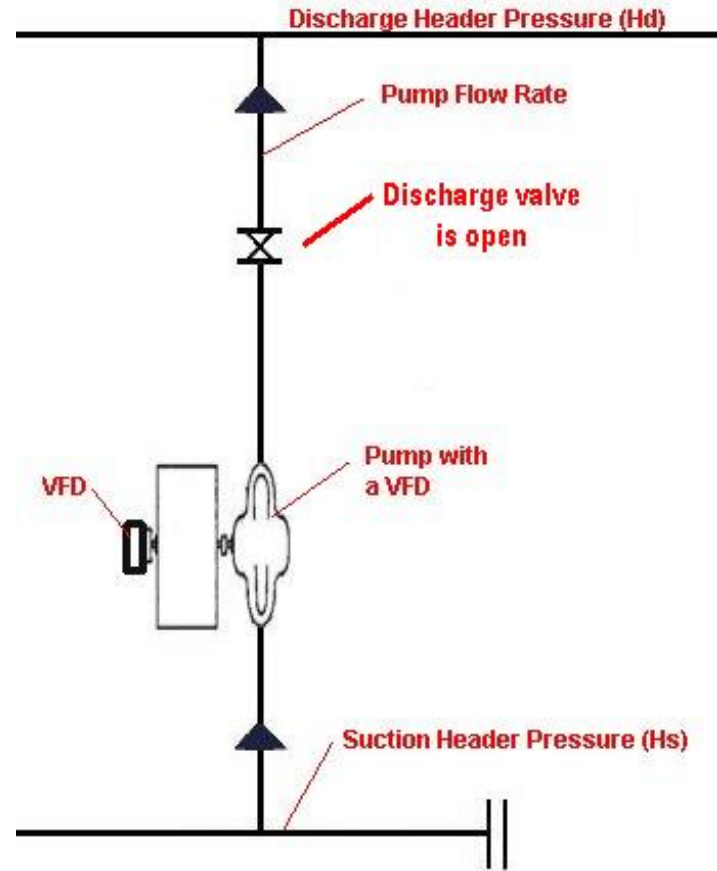
$$Q = ((H_{st} - H_{tdh}) / a)^{(1/b)}$$

Combined with Affinity Laws:

$$Q_p = F_s ((H_{stf} - (H_d - H_s) / F_s^2) / a)^{(1/b)}$$

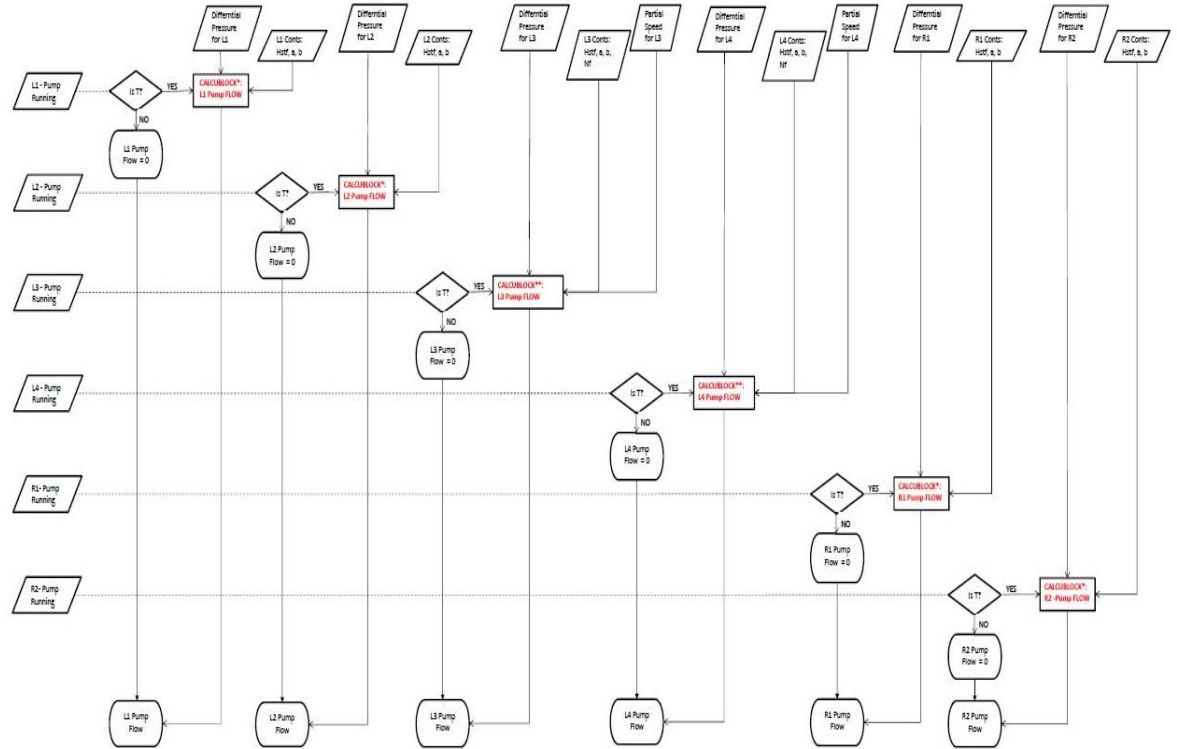
Where:

H_{stf} = pump shut-off head under full speed in feet



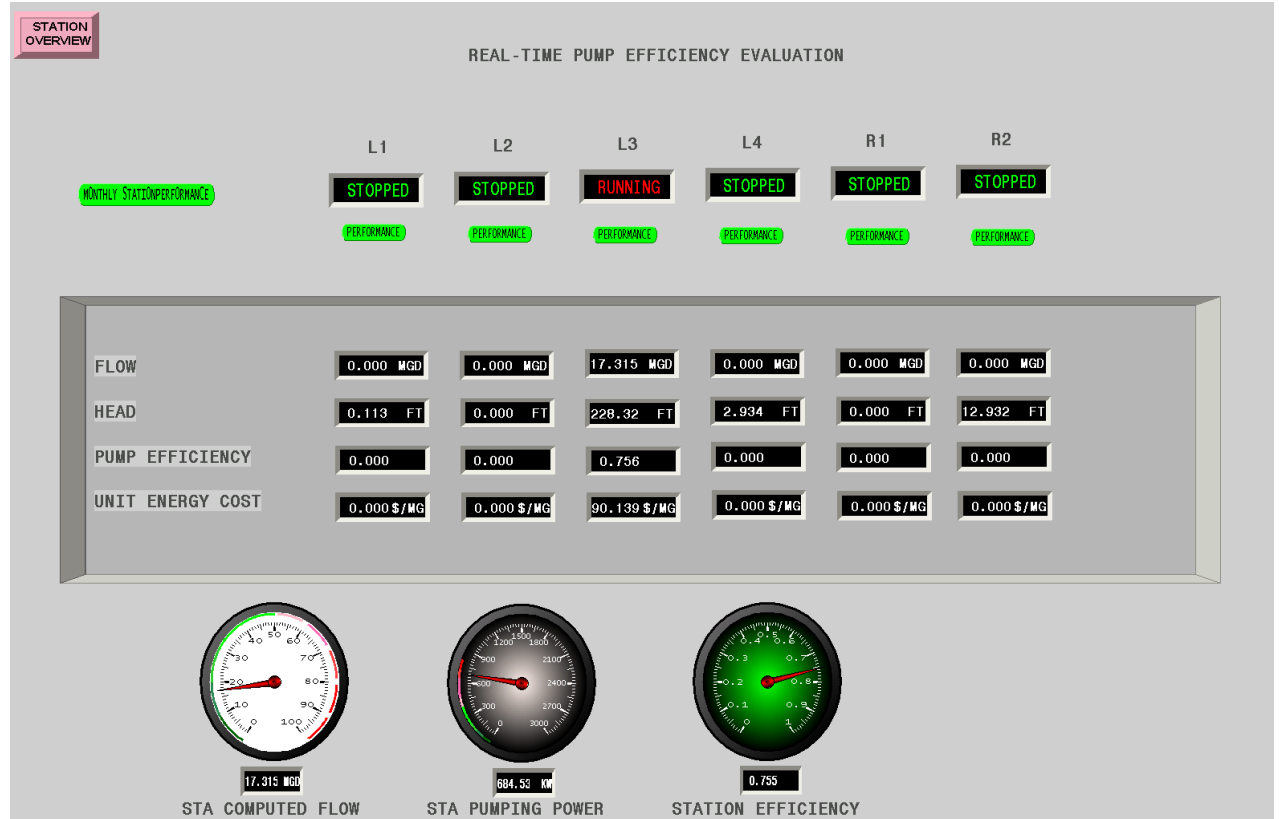
REEPS in Ovation

Flowchart for Calculating flow rates for each of the six pumps in Franklin Pump Station



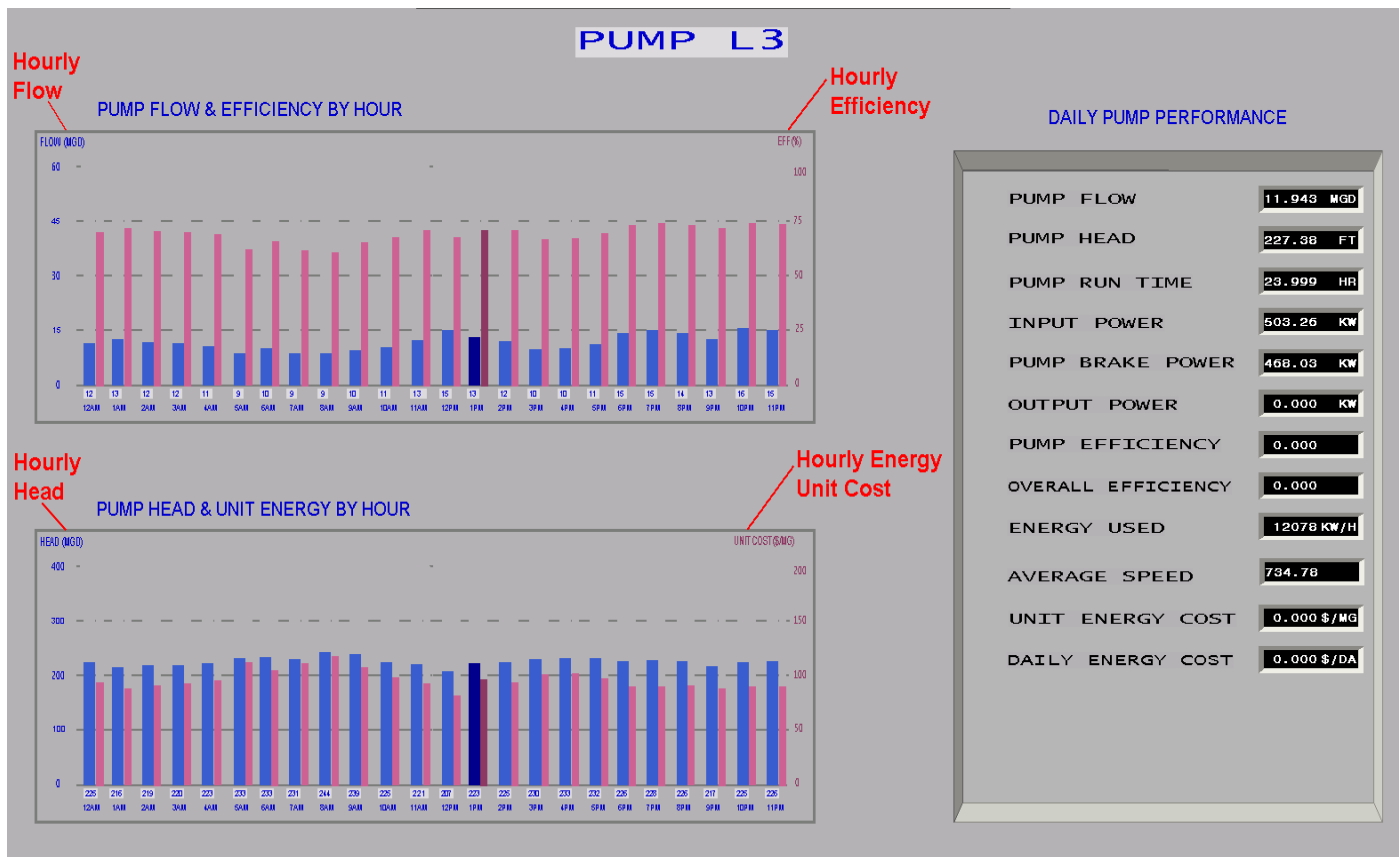
REEPS in Ovation

Real-time pump efficiency & energy cost evaluation



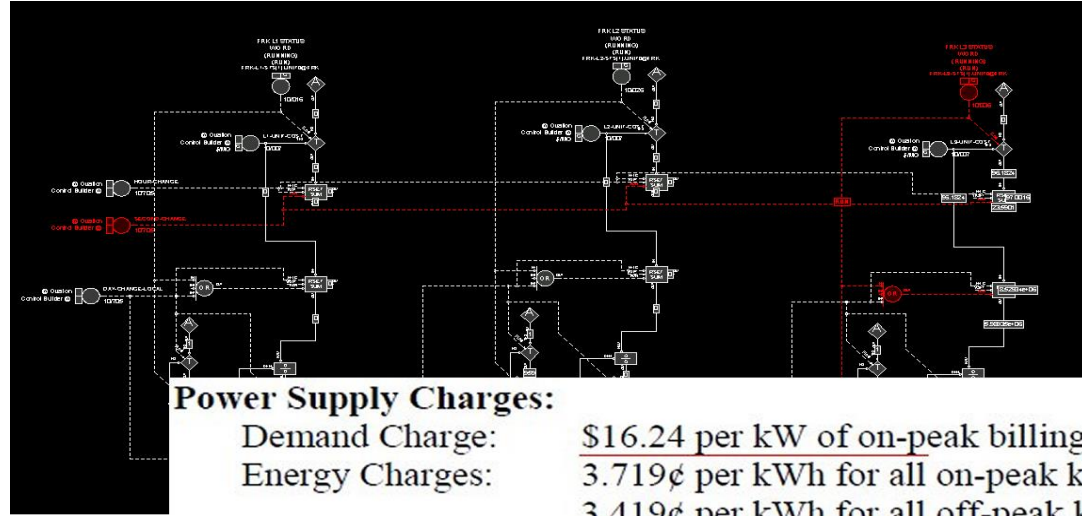
REEPS in Ovation

Hourly and daily pump performance



REEPS in Ovation

Electricity costs
are calculated
based on the
rate schedule of
DTE Energy

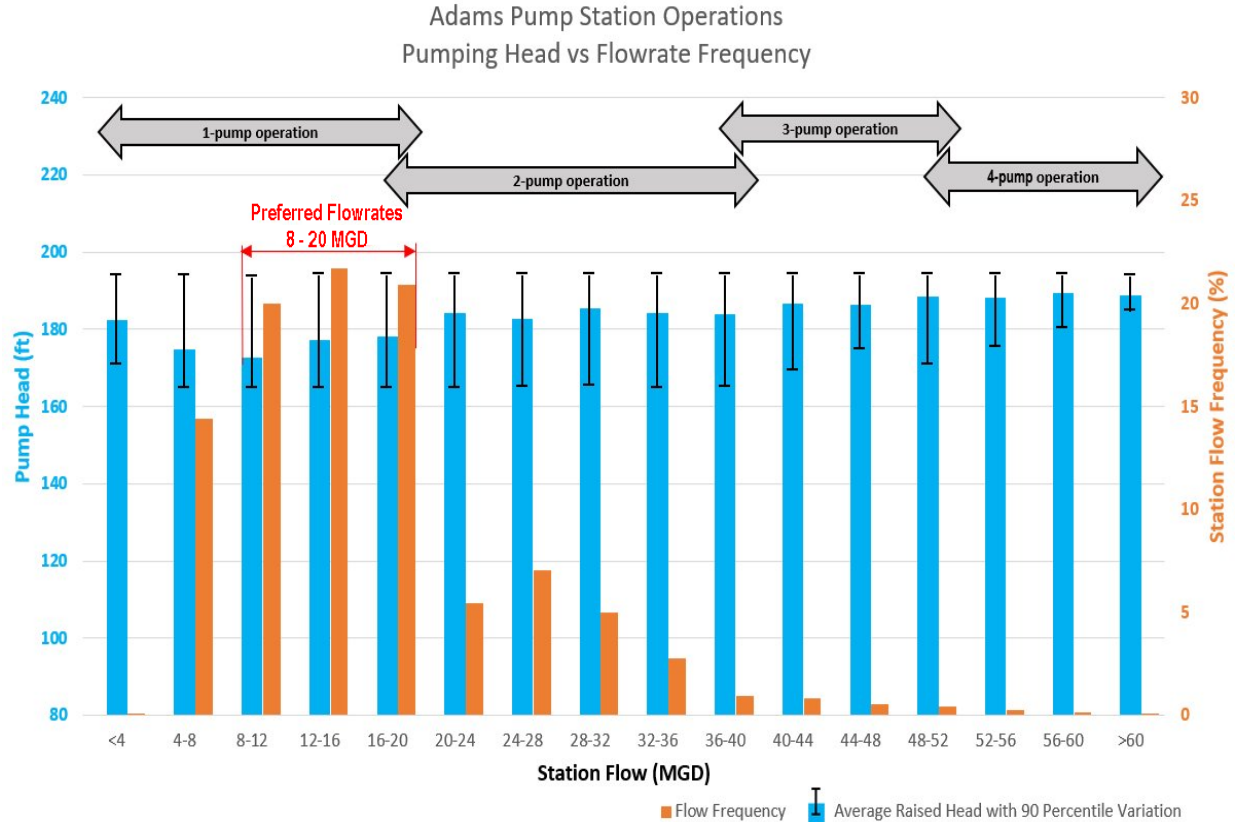


MONTHLY ON-PEAK BILLING DEMAND: The monthly on-peak billing demand shall be the single highest 30-minute integrated reading of the demand meter during the on-peak hours of the billing period.

On-peak hours are those hours between 1100 hours and 1900 hours each day, Monday through Friday, legal holidays excluded.

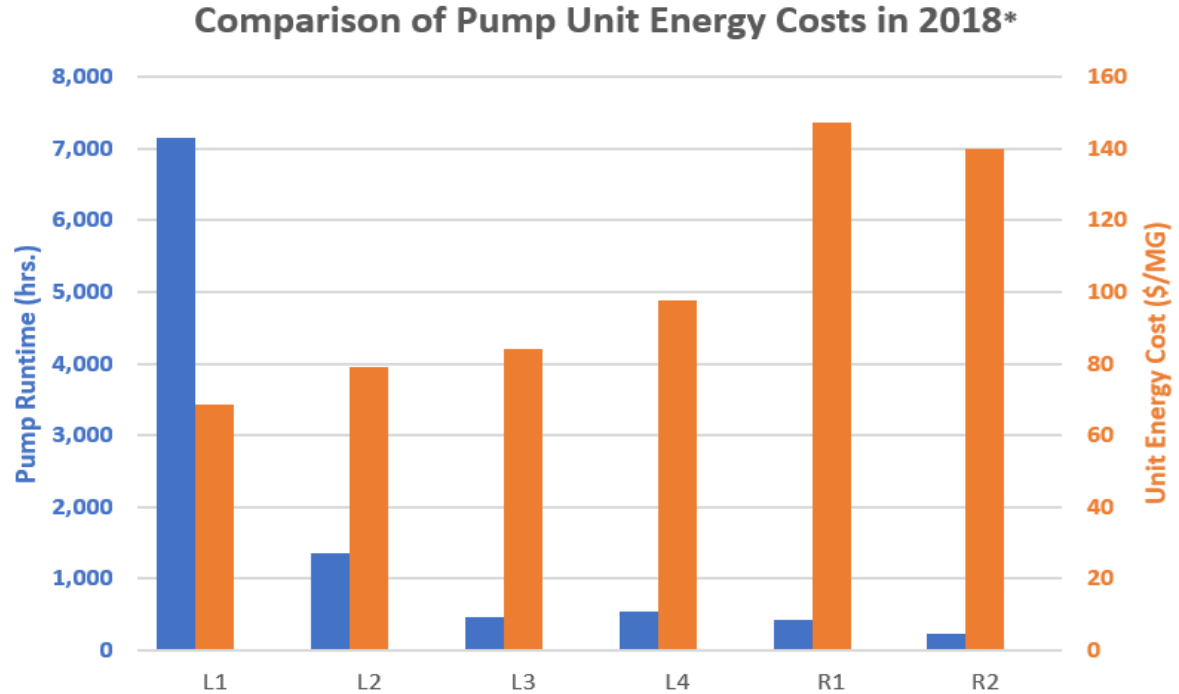
REEPS Sample Application

Pump flow, head & efficiency analysis for Adams Station



REEPS Sample Application

Pump flow, head & efficiency analysis for Adams Station



*average costs for 12 months of 2018 weighted by the monthly pump run hours; the energy costs are estimated at \$0.095 / kWh.

REEPS Sample Application

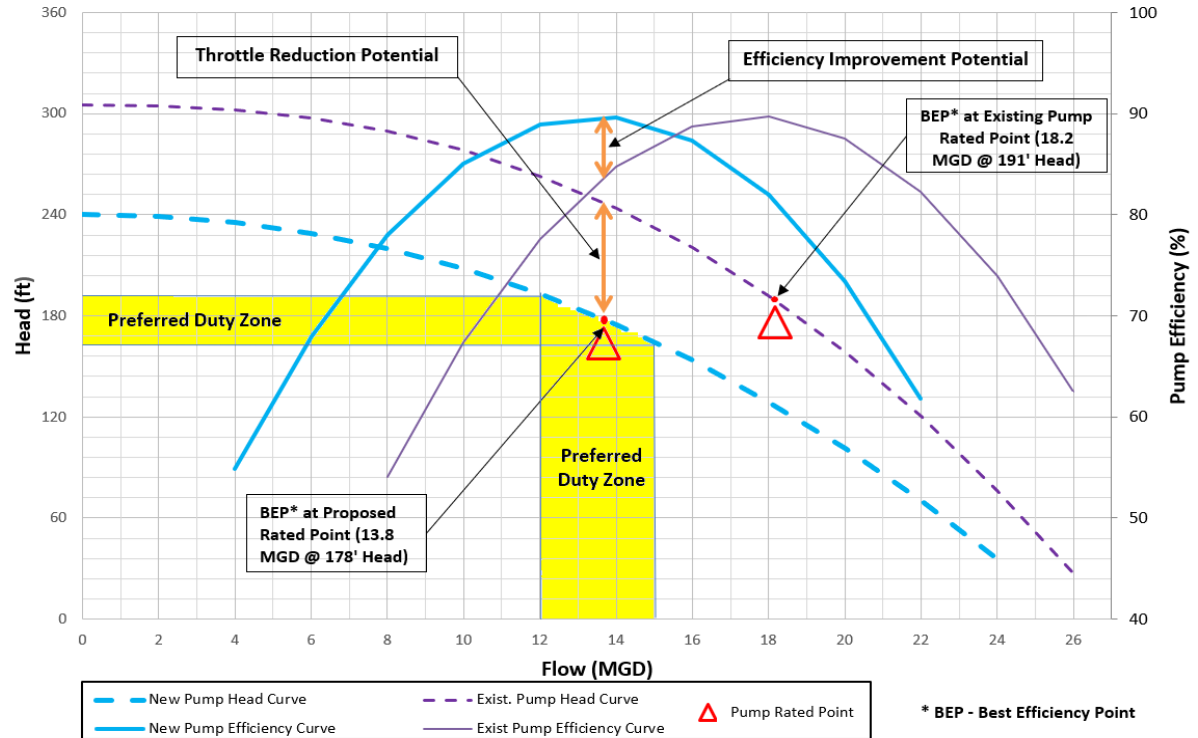
Pump flow, head & efficiency analysis for Adams Station

| Pump ID | L1 (VFD*) | L2 | L3 | L4 | R1 | R2 |
|--|-------------|--------|--------|--------|--------|--------|
| Average Pump Efficiency (%) | 66.7% | 77.4% | 77.7% | 65.7% | 81.5% | 80.6% |
| Average Discharge Valve Position (%) | ~100% | 80.4% | 54.0% | 77.4% | 23.3% | 41.8% |
| Energy Loss Due to Throttling (kWh) | No Throttle | 41,635 | 27,985 | 17,273 | 83,956 | 41,242 |
| Throttling Energy Loss (%) | No Throttle | 5.9% | 11.8% | 5.6% | 25.3% | 21.0% |
| Overall Efficiency with Throttling (%) | 64.7% | 67.6% | 62.0% | 56.8% | 52.1% | 55.5% |
| *VFD has a efficiency of 97% on average. | | | | | | |

REEPS Sample Application

Pump flow, head & efficiency analysis for Adams Station

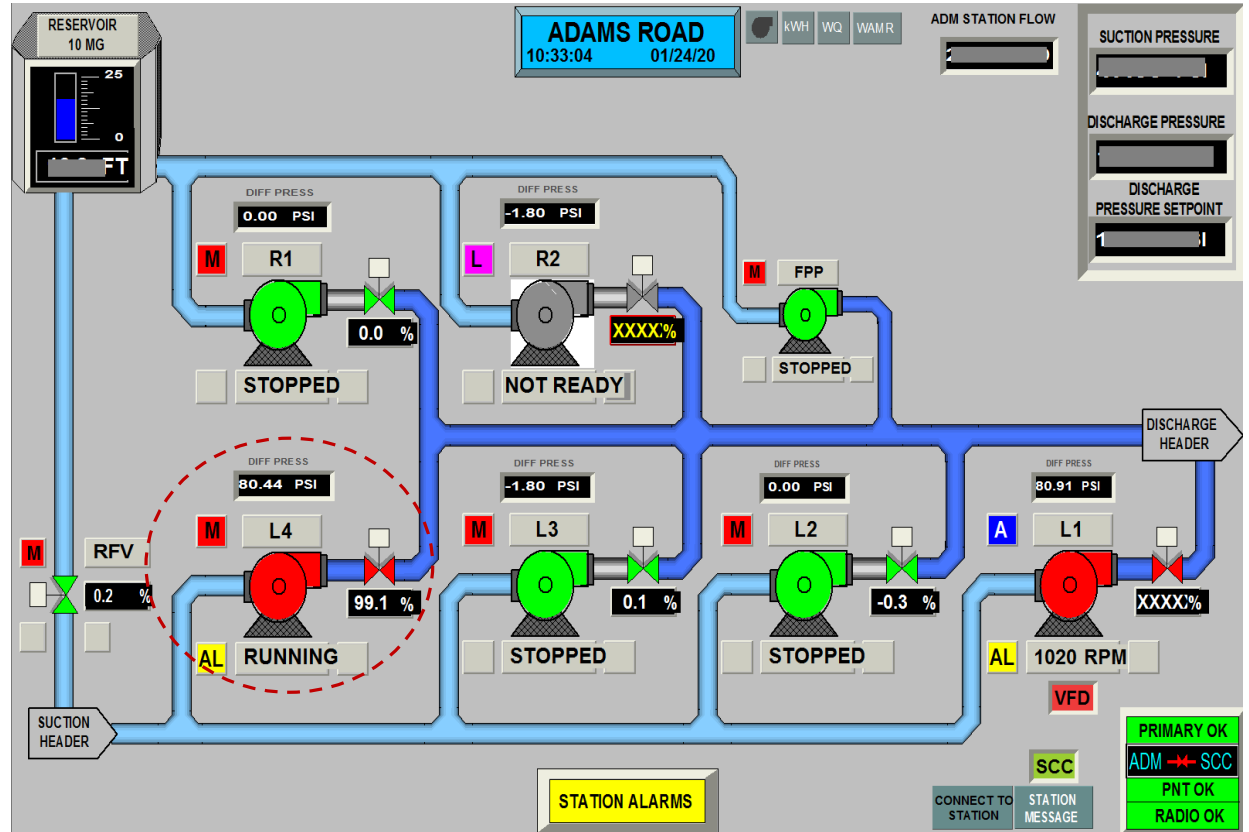
Pump L4 Re-Sizing



REEPS Sample Application

Pump flow, head & efficiency analysis for Adams Station

Using Re-Sized Pump L4



REEPS Sample Application

Pump flow, head & efficiency analysis for Adams Station

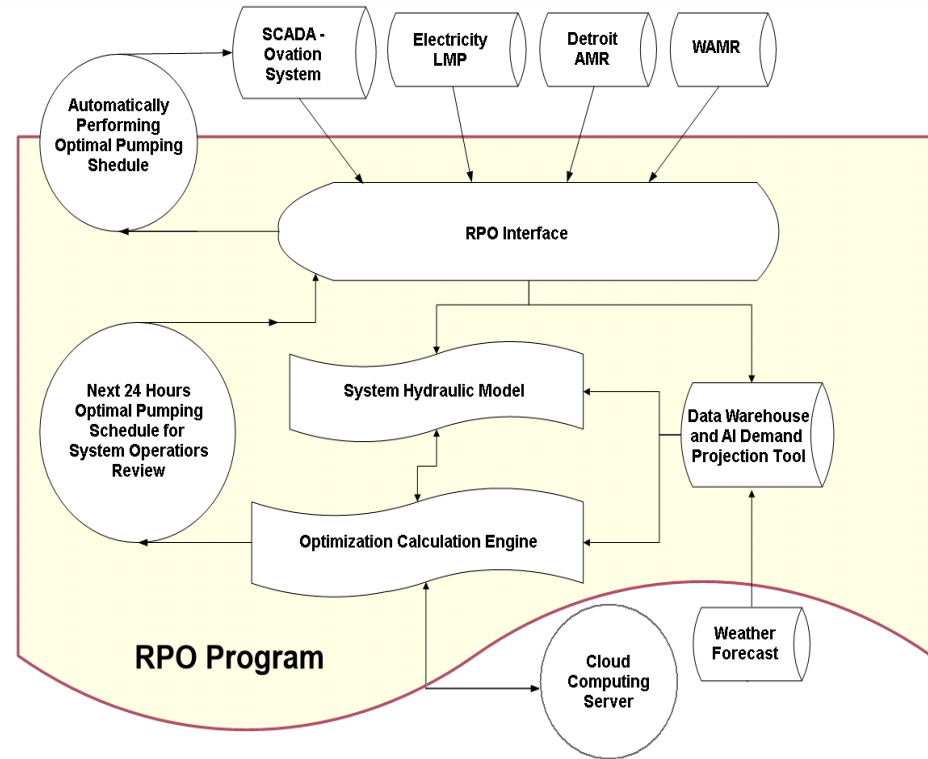
Energy Savings by Using Re-Sized Pump L4

- Pump efficiency increased from <70% to 82%
- Pump discharge throttling reduced from >50% to approximately 10%
- Annual energy savings is estimated about 180,000 kWh (approx. 6%)
- The overall unit energy cost reduced from \$81.1 per MG to \$75.7 per MG (annual cost savings of about \$16,500)

REEPS Sample Application

- Generate optimal pumping schedule for next 24-hour
- Reduce on-peak billing demand of electricity
- Reduce energy costs against DTE Energy's tariff
- Optimize water transmission paths to reduce pumping requirements
- Use cloud computing technology to speed up the calculation in optimization algorithm engine

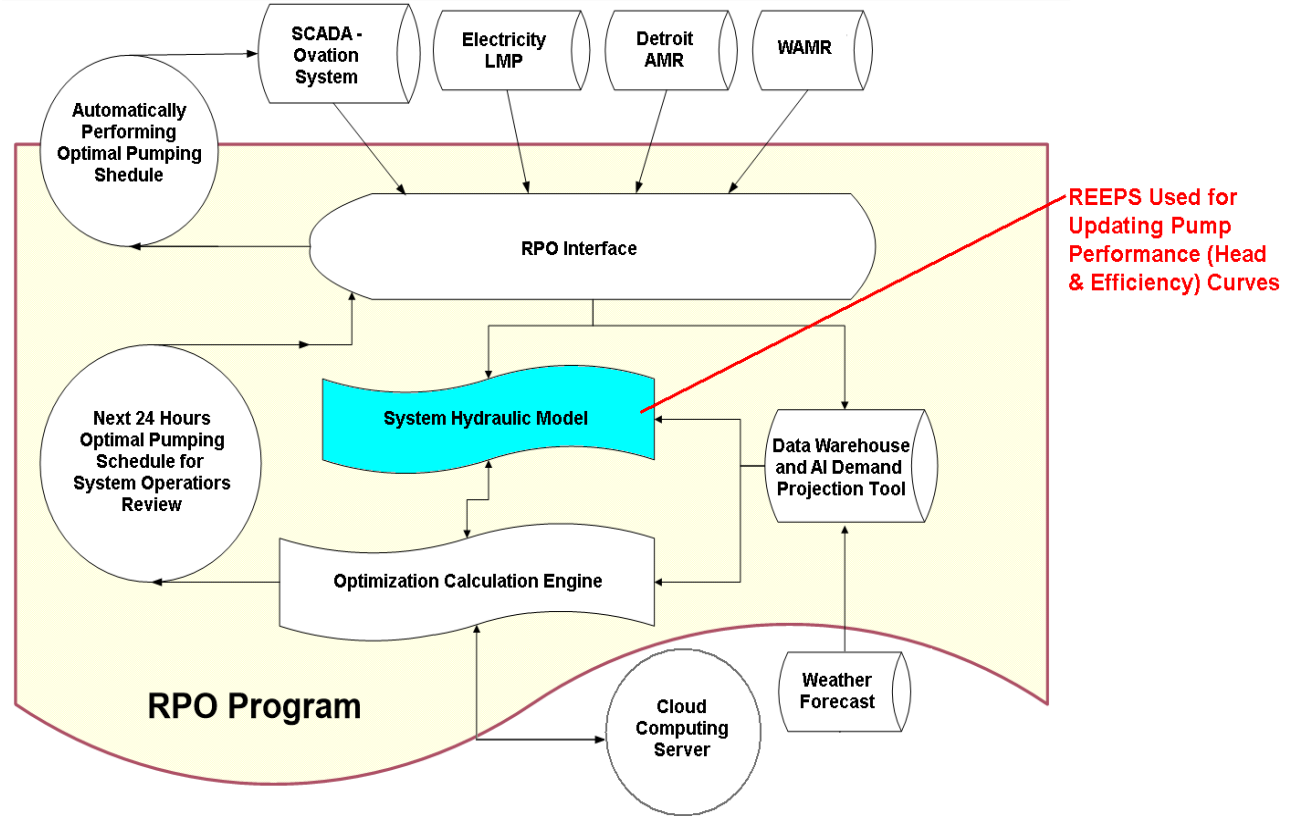
Real-time Pumping Optimization Project



REEPS Sample Application

REEPS helps
real-time
pumping
optimization
program

Real-time Pumping Optimization Project



Questions?



THANK YOU

